

Near-wall effects for momentum, heat and mass transport

in gas-particle suspensions at moderate
Reynolds numbers

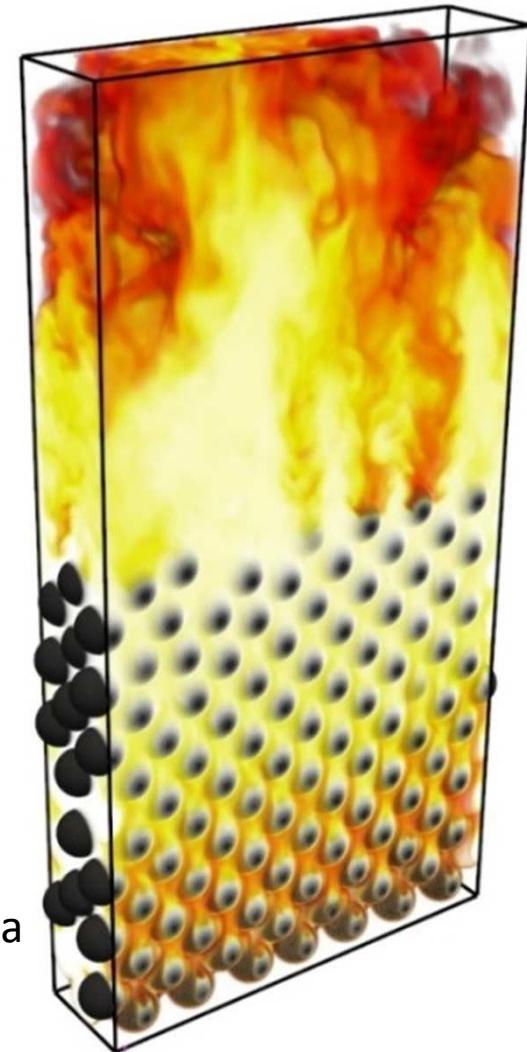
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Presented at the APS DFD Meeting 2016, Portland, OR

Motivation: engineering models

Why studying heat and mass transfer in wall bounded gas-particle suspensions ?

- **Local inhomogeneity of fluid and particle temperature** close to walls can **drastically** affect process performance. This can cause problems in chemical reactors, or heat exchangers (e.g., **local overheating, cool regions near walls**)
- Root cause are perturbations that can happen at small length scales, **below the resolution of classical models for averaged quantities** employed in engineering practice
- We expect that models based on **spatially averaged equations** require **special boundary treatment (closures)**

We use **Particle-Resolved Direct Numerical Simulation (PR-DNS)** to quantify **key effects** to lay the foundation for modeling wall effects

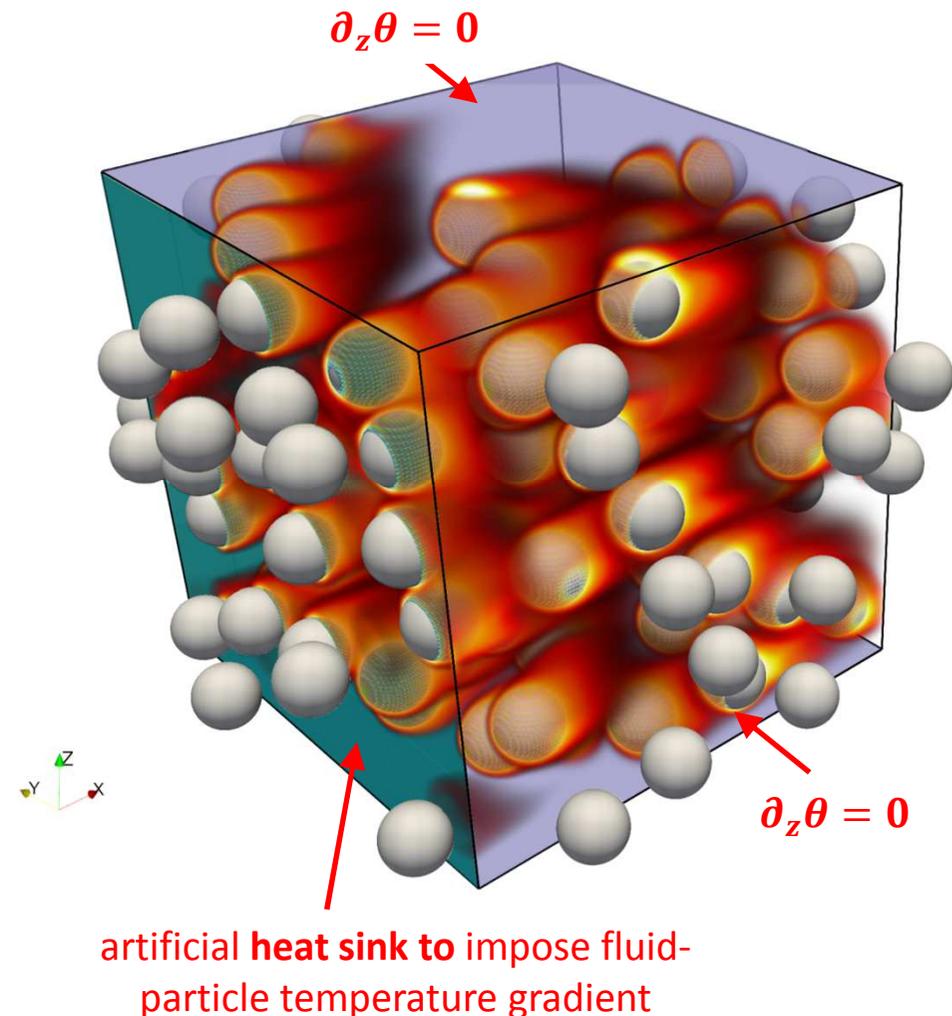
Particle-Resolved DNS

- Particle bed generated via bi-axial **compaction** in the xy plane using **LIGGGHTS**^{®1}
- Flow and temperature fields are solved in a xy **periodic domain**. Particles are isothermal
- **CFDEM**[®]**Coupling**¹ to solve the governing equations for the continuum phase
- Particles are represented by forcing terms in the governing equations (solved in a **Cartesian grid**) evaluated using the **HFD-IB**² method

¹Kloss et al., *Progress in Computational Fluid Dynamics*, 12:140–152, 2012

²Municchi et al., *CFDEMcoupling user meeting 2016*, Linz

Boundary conditions: temperature field



Current study: parameter space

- Reynolds number fixed to **100**:

$$Re = \frac{\rho_f (1 - \phi_p) u d_p}{\nu} = \frac{\rho_f U_S d_p}{\nu}$$

- Momentum and heat transfer for suspensions characterized by $\phi_p = 0.1 - 0.4$
- Multiple realizations to ensure statistical significance (>2,000 particles considered for each case)
 - $\phi_p = 0.1$: **40 realizations**
 - $\phi_p = 0.2$: **20 realizations**
 - $\phi_p = 0.3$: **20 realizations**
 - $\phi_p = 0.4$: **16 realizations**

ρ_f : fluid phase density

ϕ_p : particle volume fraction

d_p : particle diameter

ν : fluid viscosity

U : reference fluid velocity

U_S : superficial velocity

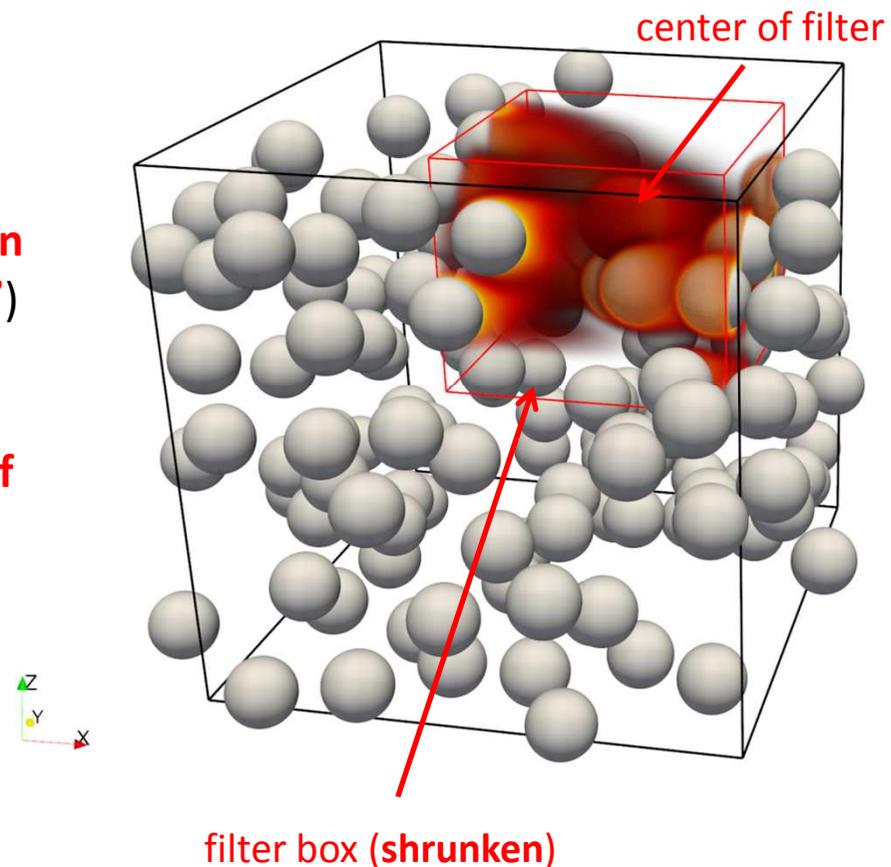
Particle-based quantities: filtering

- We make use of the filtering toolbox **CPPPO**³ to spatially average (“filter”) the continuum phase properties around **each particle**

$$\varrho = \frac{L_{filter}}{d_p} \quad \begin{array}{l} \text{Dimensionless} \\ \text{filter size} \end{array}$$

- CPPPO is also employed to draw more “conventional” statistics (e.g., **profiles in wall-normal direction, “pancake filter”**)
- Filter boxes are **shrunk in the vicinity of wall boundaries**, same as done for wall bounded single phase turbulent flow⁴

Lagrangian filtering: wall particles



³Municchi et al., *Computer Physics Communications*, 2016 207:400-414

⁴Sagaut, 2006, Springer

Particle-based quantities: filtering

Definitions

$$\mathbf{f}_i^d \equiv \mathbf{f}_i - \mathbf{f}_i^{\nabla p^e}$$

- \mathbf{f}_i : Total force acting on particle i
- \mathbf{f}_i^d : Drag force acting on particle i
- $\mathbf{f}_i^{\nabla p^e}$: Force due to mean pressure gradient **on particle i**
- ∇p_i^e : **Mean pressure gradient seen by particle i**

$$F_i \equiv \frac{\mathbf{f}_{i,x}^d}{3 \pi \mu d_p (1 - \phi_i^e) \mathbf{u}_{i,x}^e}$$

- d_p : Diameter of particles
- F_i : Drag coefficient of particle i
- ϕ_i^e : Volume fraction of solid phase around particle i .
- $\mathbf{u}_{i,x}^e$: Normalized mean **relative velocity seen by particle i**

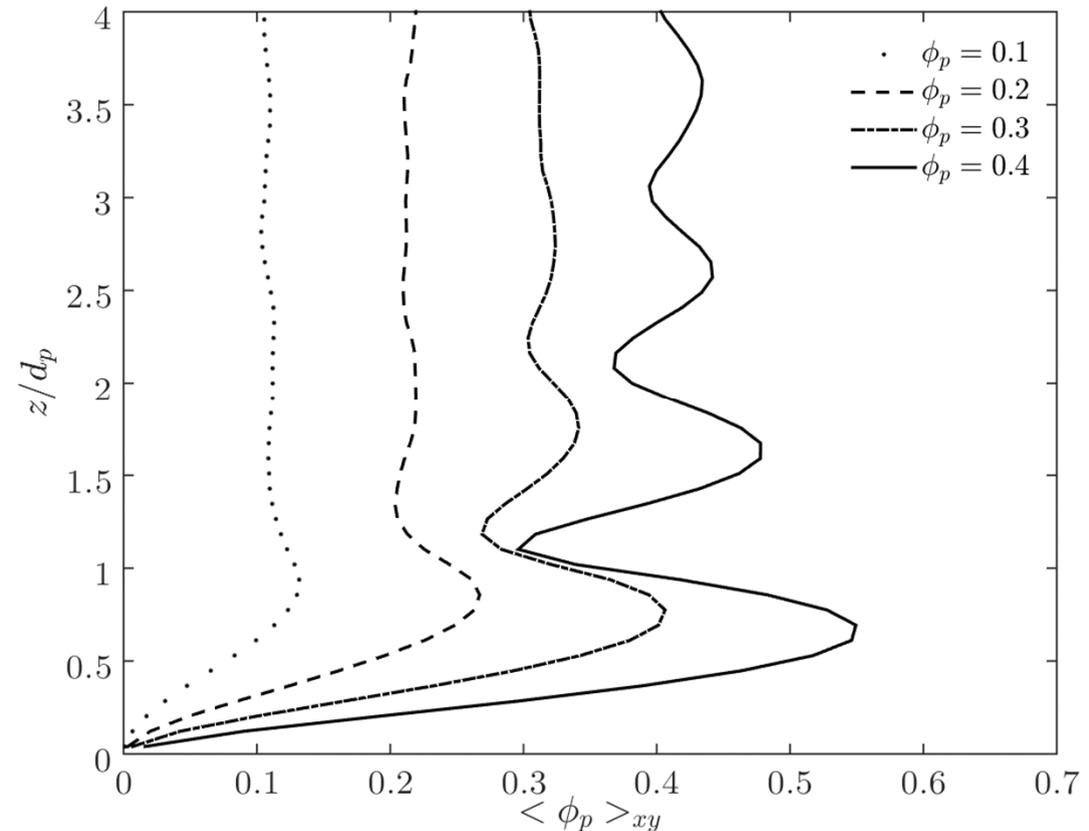
$$Nu_i \equiv \frac{Q_i^* Pe}{\pi (\theta_s - \theta_i^e)}$$

- Q_i^* : Normalized heat exchange rate
- Pe : Peclet number
- θ_i^e : Fluid **temp. seen by particle i**
- θ_s : Surface temperature of particle i

Wall effects: particle distribution

- The particle volume fraction is **inhomogeneous** on the scale of d_p
- This inhomogeneity results also in the continuum phase properties to be inhomogeneous **on the same scale**

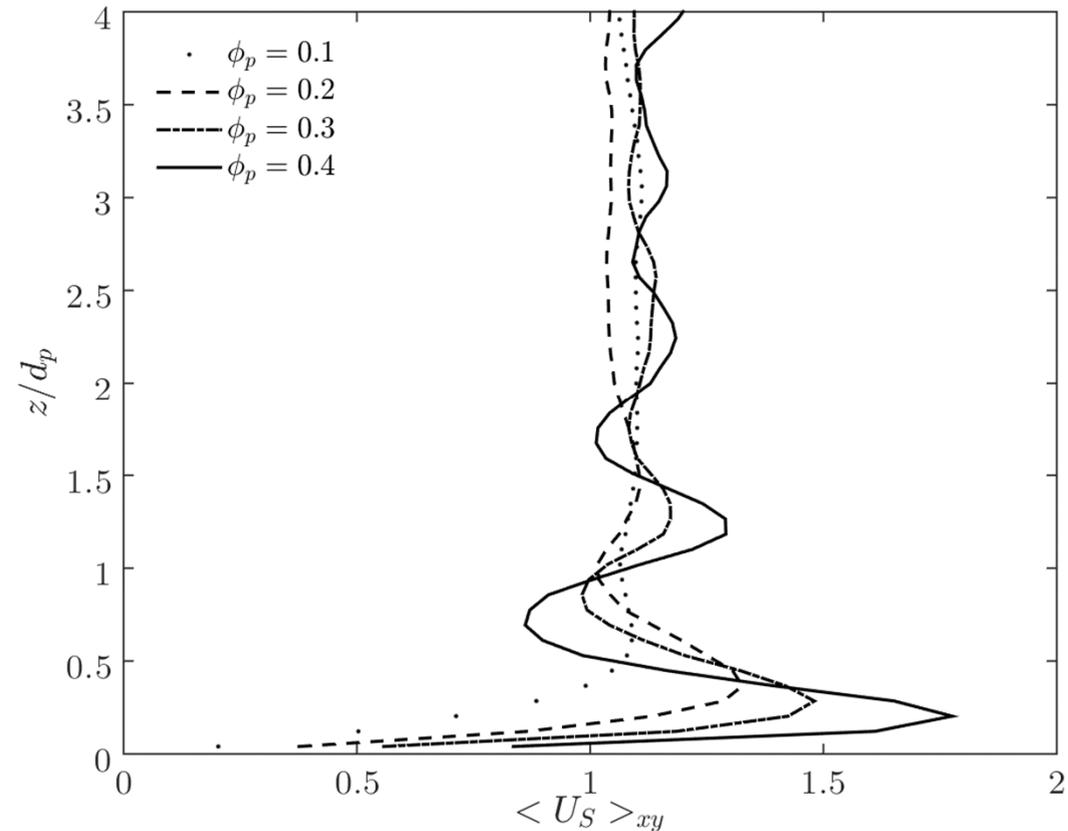
“pancake” filter



Wall effects: temperature and velocity

- The temperature profile has a **minimum at the wall** due to the low value of ϕ_p and thus, the reduced exchange surface
- In addition, the exchanged heat is quickly **removed by convection**
- As ϕ_p is **increased** the fluid flows through the path of **minimum resistance: the wall neighborhood**

“pancake” filter



U_S is calculated using the global ϕ_p and the local U .

Wall effects: particle-based Drag

- $\phi_p = 0.1$

The drag is **larger at the wall** and **larger for low ϱ**

- $\phi_p = 0.2$

Oscillations close to the wall and weak dependence on ϱ

- $\phi_p = 0.3$

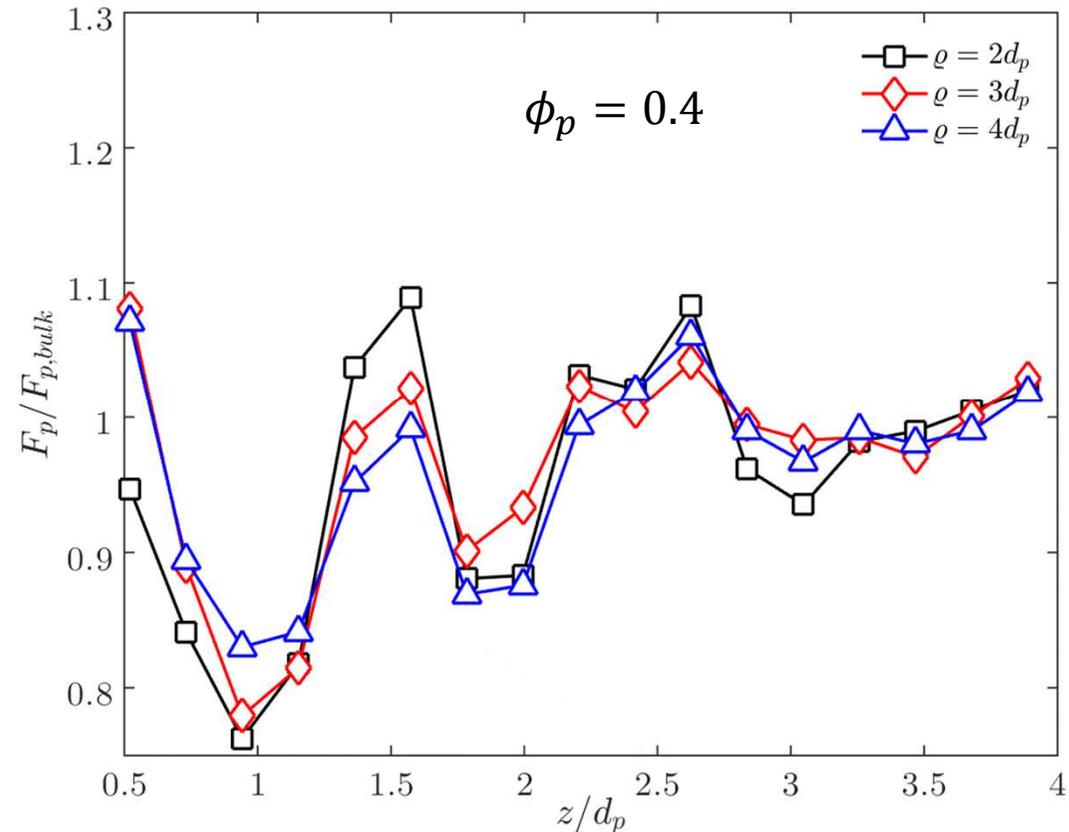
Minimum at $z = d_p$ and **no dependence** on ϱ

- $\phi_p = 0.4$

The drag is **smaller at the wall** and **smaller for ϱ low**

Oscillations propagate up to $z = 2.5d_p$

“box” filter



The drag coefficient near the wall tends to **decrease with increasing ϕ_p**

Wall effects: particle-based Nusselt number

- $\phi_p = 0.1$

No large wall effects

- $\phi_p = 0.2$

Little changes

- $\phi_p = 0.3$

Weak dependency on ϱ

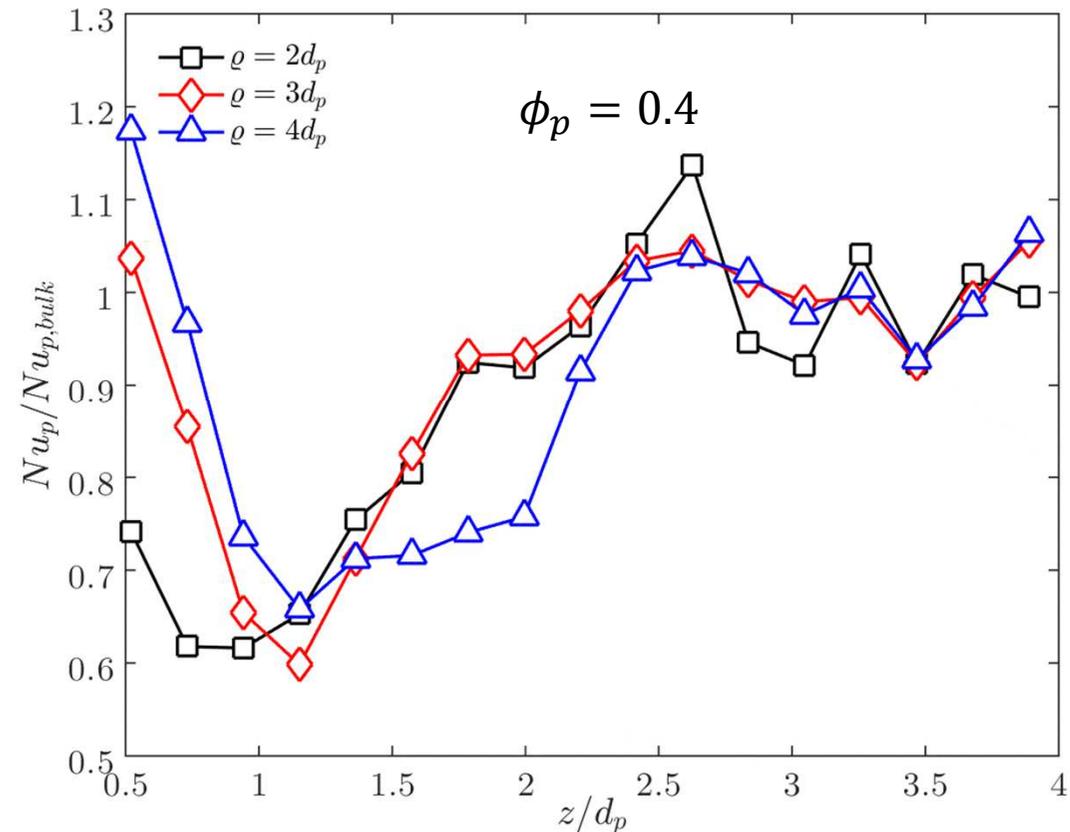
Minimum at $z = d_p$

- $\phi_p = 0.4$

Marked dependency on ϱ

Stronger minimum and effects up to $z = 2.5d_p$

“box” filter



Similar behavior as for the drag coefficient but increased Nu at the wall with increasing ϕ_p

Can we exploit an analogy?

Conclusions

Which situations are affected by wall effects ?

...for the moderate Reynolds number studied here...

- **Always for the drag coefficient.** Not taking into account wall effects can lead to an error up to **40%**. **Wall effects are stronger for increasing values of ϕ_p**
- In case **$\phi_p > 0.2$ also for the Nusselt number** . The particle-fluid heat transfer does not seem to be significantly affected in dilute suspensions in case of **adiabatic walls**. In dense suspensions we observe **deviations of -40% to $+20\%$** for the per-particle Nusselt number

What is the key consequence of wall effects ?

- Even when using **adiabatic walls, extreme fluid temperature gradients** near the wall are observed

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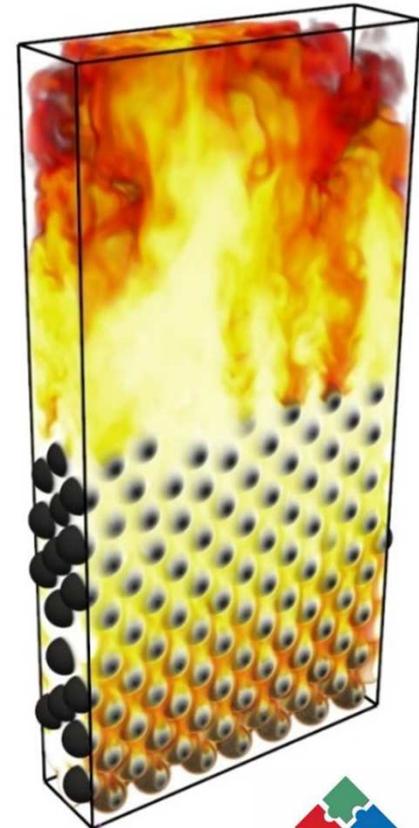
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Funding was provided by **EC Grant #604656** (software development), and “**NAWI Graz**” (access to VSC-3 and dcluster.tugraz.at).

Code available via github.com/CFDEMproject

For more information search “Radl” on online.tugraz.at



NanoSim

